

Transverse Plane-Wave Analysis of Short Elliptical End-Chamber and Expansion-Chamber Mufflers

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The flow-reversal end chambers are used quite often in commercial automotive mufflers. The conventional axial plane-wave theory is not able to predict their acoustic performance because of the fact that the length of the end chambers is not enough for the evanescent three-dimensional modes generated at the junctions to decay sufficiently for frequencies below the cut-off frequency. Also, due to the large expansion ratio at the inlet, the first few higher-order modes get cut on even in the low-frequency regime. This necessitates a finite element or boundary element analysis, which is cumbersome and time-consuming. Therefore, an ingenious one-dimensional method has been developed. It models plane-wave propagation in the transverse direction between the incoming pipe and the return pipe, with the lateral-end cavities being modeled as variable-area quarter-wave resonators. Making use of this novel approach, the transfer matrices have been derived for elliptical and circular cross-section mufflers, which enable these elements to be analyzed along with the rest of the muffler elements by means of the transfer matrix-based muffler program. Through a comparison with a full, three-dimensional analysis on commercial software, it is shown that the one-dimensional approach presented in this paper is able to predict the transmission loss quite accurately up to about 1000 Hz for typical automotive mufflers.

NOMENCLATURE

c_0	– sound speed in the medium		
d_1, d_2	– diameters of the inlet and outlet pipes	$S(x)$	– cross-sectional area of muffler as a function of distance x , measured along the plane-wave propagation path
d_3	– diameter of the pass tube in the flow-reversal muffler	S_u, S_d	– cross-section area of the inlet and outlet pipe, respectively
D_1, D_2	– major axis and minor axis of the elliptical muffler	S_1, S_2	– cross-section area of the muffler at the inlet and outlet, respectively, perpendicular to the assumed plane-wave path
D	– diameter of the circular expansion chamber	$u(x)$	– acoustic particle velocity along the direction of the plane-wave propagation at any generic point in the muffler system
j	– $(-1)^{1/2}$	U_i, U_o	– mean-flow velocity at the inlet and outlet pipe, respectively
k_0	– the wave number, ω/c_0	U_1, U_2	– mean-flow velocity at the muffler inlet and outlet, respectively
K_{1u}, K_{2u}, K_{3u}	– stagnation pressure-loss coefficients at the inlet junction	v_u, v_d	– acoustic-mass velocity at the upstream point and downstream point, respectively
K_{1d}, K_{2d}, K_{3d}	– stagnation pressure-loss coefficients at the outlet junction	v_{r1}, v_{r2}	– acoustic-mass velocity at the quarter-wave resonators at inlet and outlet junctions, respectively
L	– axial length of the muffler	v_1, v_2	– acoustic-mass velocity at the inlet and outlet junctions in the muffler at the beginning and end of the transverse plane path, respectively, for a short-length muffler with a staggered configuration
M_d, M_u	– Mach number at the outlet (U_o/c_0) and inlet (U_i/c_0), respectively	Y_d	– characteristic impedance at downstream point in the outlet pipe, c_0/S_d
M_1, M_2	– mean-flow Mach number at the muffler inlet (U_1/c_0) and outlet (U_2/c_0), respectively, perpendicular to the assumed plane-wave path	Y_u	– characteristic impedance at the upstream point in the inlet pipe, c_0/S_u
n	– total number of small parts over which the region of interest is divided	Y_1, Y_2	– characteristic impedance at the muffler inlet (c_0/S_1) and outlet (c_0/S_2), respectively, perpendicular to the assumed plane-wave path
$p(x)$	– acoustic pressure field at any generic point in the muffler system		
p_u, p_d	– acoustic pressure at the upstream inlet pipe and point in the downstream outlet pipe, respectively		
p_{r1}, p_{r2}	– acoustic pressure at the inlet and outlet junctions of the quarter-wave resonators, respectively		
p_1, p_2	– acoustic pressure at the inlet and outlet junctions in the muffler at the		