
A New Hybrid Approach for the Thermo-acoustic Modelling of Engine Exhaust Systems

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The time-domain method of characteristics and the frequency-domain linear acoustics method are the two different methods of thermo-acoustic modelling of engine exhaust systems in order to predict exhaust noise. To overcome the disadvantages of both the time-domain and the frequency-domain approaches, a hybrid approach has been developed which couples the acoustical description of the muffler piping system to the acoustic source more realistically than the usual time invariant linear model. A time domain model and the corresponding frequency domain model of a linear dynamical system are interrelated by the Fourier transform pair. In the present work, the cylinder/cavity is analysed in the time domain to calculate exhaust mass flux history at the exhaust valve by means of the method of characteristics, solving a number of equations simultaneously at the valve junction. This analysis has been done by making use of an interrelationship between progressive wave variables of linear acoustic theory and those of the method of characteristics. In this approach, nonlinear propagation in the exhaust pipe is neglected and radiation impedance at the end of the exhaust tail pipe is duly taken into account. Apart from this, actual reflection of the forward wave due to the presence of muffler is incorporated to make the analysis more realistic. Damping effects present in the fluid have also been taken into account. Computational results have been corroborated by experimental data for a single-cylinder, four-stroke cycle diesel engine.

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Nomenclature

A	– Nondimensionalised speed of sound = a/a_{ref} , magnitude of the forward wave	v	– Acoustic mass velocity
a	– Sound speed	W	– Acoustic power
B	– Reflected part of the forward moving pressure wave	X	– Nondimensionalised space variable
C_p	– Specific heat at constant pressure	x	– Space coordinate
C_v	– Specific heat at constant volume	Y	– Characteristic impedance of a pipe
D	– Diameter	Z	– Nondimensionalised time variable; impedance
E	– Polynomial coefficient for the exhaust area	z	– Space coordinate
I	– Polynomial coefficient for the inlet area	γ	– Ratio of specific heats
IL	– Insertion loss	θ	– Crank angle
j	– $\sqrt{-1}$	ρ	– Density
k	– Wave number	ω	– Circular frequency
L	– Reference length	L_c	– Connecting rod length
L_w	– Acoustic power level	RBP	– Rated Brake Power
l	– Muffler element length	CR	– Compression Ratio
M	– Mean flow Mach number		
MOC-2	– Two characteristics approach to the method of characteristics		
MOC-3	– Three characteristics approach to the method of characteristics		
\dot{m}	– Mass flow rate		
P	– Forward Riemann variable		
Q	– Rearward Riemann variable		
p	– Acoustic pressure		
R	– Gas constant		
S	– Area of cross-section of a pipe; acoustic state vector		
t	– Time variable		
U	– Mean flow velocity		
u	– Acoustic particle velocity		
VR	– Velocity ratio		

1. INTRODUCTION

Noise pollution from automobiles has become a serious problem. Hence, the design and analysis of mufflers for engine exhaust systems are active areas of research. In order to create a suitable design for a muffler, the noise generating source has to be analysed integrally with the muffler. Generally, mufflers are modelled in the frequency domain, making use of the transfer matrix method.¹ This method requires a prior knowledge of the source characteristics as a function of frequency, which is still a challenge. In fact, in view of the strong time variance and nonlinearity of engine exhaust sources, there are serious doubts about the existence of unique source characteristics.² Such a source, with varying mass flows through the valves, can be dealt with best in the time domain. The time domain analysis, based on the method of characteristics, is complete in itself but is very cumbersome.